

PHEN 612

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WEEK 5

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Chapter 8

Steady-state Non-isothermal Reactor Design

8.1 In addition to the Arrhenius equation, we need to understand how the temperature affects the conversion X .

8.2 Energy Balance

- First law (closed system): $d\hat{E} = \delta Q - \delta W$
- First law (open system): $\frac{d\hat{E}_{sys}}{dt} = \dot{Q} - \dot{W} + F_{in}E_{in} - F_{out}E_{out}$
- The first law for open systems can be extended to several inlet and outlet flows.
- These equations help to understand the influence of heat on a system operating isothermally

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8.2.2 The work term (\dot{W}) accounts for flow work (due to materials crossing the boundary) and shaft work (e.g., stirrer in a CSTR, turbine in a PFR):

$$\dot{W} = \sum_{i=1}^n F_i P \tilde{V}_i \Big|_{out} - \sum_{i=1}^n F_i P \tilde{V}_i \Big|_{in} + \dot{W}_s$$

By using $H_i = U_i + P \tilde{V}_i$ and neglecting the contribution of potential and internal energies

$$\frac{d\hat{E}_{sys}}{dt} = \dot{Q} - \dot{W}_s + \sum_{i=1}^n F_{i0} H_{i0} - \sum_{i=1}^n F_i H_i$$

- F: is the molar flow rate; H: specific molar enthalpy; V: specific molar volume; P: the pressure; U: internal energy

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8.2.3 We need to develop relationship between system temperature, conversion, molar flow rate and system parameters.

8.2.4 We know that heat of reaction plays a significant role in reaction engineering. How do we integrate the heat of reaction in the heat transfer equation?

Steady-state consideration: accumulation rate = 0

$$0 = \dot{Q} - \dot{W}_s + \sum_{i=1}^n F_{i0} H_{i0} - \sum_{i=1}^n F_i H_i$$

The reaction is given by: $A + (b/a)B \rightarrow (c/a)C + (d/a)D$

Using: $\sum_{i=1}^n F_i H_i = F_A H_A + F_B H_B + F_C H_C + F_D H_D + F_I H_I$ for the inlet and outlet flows

and $F_i = F_{A0} (\Theta_i + \nu_i X)$ we have

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8.2.5

$$\dot{Q} - \dot{W}_s + F_{A0} \sum_{i=1}^n \Theta_i (H_{i0} - H_i) - \Delta H_{RX} \langle T \rangle F_{A0} X = 0$$

with the heat of reaction defined as:

$$\Delta H_{RX} \langle T \rangle = \frac{d}{a} H_D \langle T \rangle + \frac{c}{a} H_C \langle T \rangle - \frac{b}{a} H_B \langle T \rangle - H_A \langle T \rangle$$

For a process with no phase change and constant heat capacity C_p , we have

$$H_i - H_{i0} = C_{pi} (T - T_{i0})$$

As a result:

$$\dot{Q} - \dot{W}_s - F_{A0} \sum_{i=1}^n \Theta_i C_{pi} (T_i - T_{i0}) - \Delta H_{RX} \langle T \rangle F_{A0} X = 0$$

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8.2.6

- If we use a reference condition (R):

$$\Delta H_{RX} \langle T \rangle = \Delta H_{RX}^0 \langle T_R \rangle + \Delta C_p (T - T_R)$$

we have (for no shaft work):

$$\dot{Q} - F_{A0} \sum_{i=1}^n \Theta_i C_{pi} (T_i - T_{i0}) - \left[\Delta H_{RX}^0 \langle T_R \rangle + \Delta C_p (T - T_R) \right] F_{A0} X = 0$$

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8.4 Steady-state tubular reactor with heat exchanger.

In this case:

$$\Delta \dot{Q} = U \Delta A (T_a - T) = Ua (T_a - T) \Delta V$$

- Where T_a is the ambient temperature, T , the reactor temperature, and U , the overall heat transfer coefficient. The heat exchanger area is give by a : $a = A/V = 4/D$
- With no shaft work and taking into account the design equation:

$$\frac{dF_i}{dV} = r_i = \nu_i (-r_A)$$

We have:

$$\frac{dT}{dV} = \frac{r_A \Delta H_{RX} - Ua (T_a - T)}{\sum F_i C_{pi}}$$

Heat generated
Heat removed

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In terms of conversion X

$$\frac{dT}{dV} = \frac{r_A \Delta H_{RX} - Ua(T_a - T)}{F_{A0} \left(\sum \Theta_i C_{pi} + \Delta C_p X \right)}$$

- For packed-bed:

$$\frac{dT}{dW} = \frac{r_A' \Delta H_{RX} - \frac{Ua}{\rho_b} (T_a - T)}{\sum F_i C_{pi}}$$

8.4.2 Balance on the coolant heat transfer fluid (c)

Co-current flow:
$$\frac{dT_a}{dV} = \frac{Ua(T - T_a)}{\dot{m}_c C_{Pc}}$$

Counter-current flow:
$$\frac{dT_a}{dV} = \frac{Ua(T_a - T)}{\dot{m}_c C_{Pc}}$$