Collection of typographic errors:
Design of Machine Elements by Spotts, Shoup and Hornberger,
8th edition, Prentice Hall

p.25 Eq. 5: the last term in the equation should be $\int ydA$ not $\int yda$.

Figure 1-4. sub figure (b): all y_1, y_2, y_3 should be $\bar{y}_1, \bar{y}_2, \bar{y}_3$, to conform with Eq.(6).

p. 27 Fig.1.5: the length CD should be $vd\phi$ not just v .

p. 50 Figure 1-20, subfigure (b), there is letter "T" under dx. It should be a Greek letter τ

p. 59 Above Eq. (33) A = Area of cross section. A is missing.

p. 63: Above Eq. (39), It should be $\sin^2\varphi$, not $\sin^{2\varphi}$, in the first formula.

p. 65 Line 2 from top, σ_x should be replaced by σ_y

Both Eq. (41) & (42): Inside the square root the σ_2 should be $\sigma_{x.\text{ (Two places)}}$

p. 66 Eq. (44) , the first term under the square root is $(\sigma_x-\sigma_y/2)$, the x was a mistake.

p. 67 Figure 1-32 sub figure (a) The τ_{xy} = -5000 psi should be τ_{xy} = 5000 psi the minus sign is wrong.

Figure 1-32: Three lines from the bottom line, (e) The maximum shear stress τ_{max} and the corresponding…. The Z_{max} was a mistake.

p. 73: The CD ROM Module 1-5 output should read as:
S_1 = 33.7 M Pa
S_2 = 19.1 M Pa
S_3 = 7.1 M Pa.
Principal stresses are defined as "S" on p.66 and 67. S_1 is always the largest algebraic number. $S_1 > S_2 > S_3$

p. 92: **Problem 40.** Answers are τ = 2.36, 4.05, 5.06, and 5.4 MPa, NOT τ = 2.1, 3.6, 4.5, and 4.8 MPa

p. 104: **Problem 1.87**, ID = 4.03 in., not 4.3 in.

p. 104: **Problem1-88**: Answers: Max. = -14 MPa, Min. = -84 MPa, Max. Shear = 35 MPa

p. 105: **Problem 92**, σy = -14 MPa. The minus sign is missing.

p 109: **Problem 106**: 12 in. length should be read as 300 mm, and length becomes 300.15 mm not 200.15 mm

p.135: S_1=145.21, S_2 = 53.37, S_3 = 41.42

p. 170: **Problem 2-1**: Material is 1035 hot rolled steel, with 1 inch width.

p. 170: **Problem 2-3**: Answers: (a) Fs = 1.39, (b) Fs = 1.46

p. 171: **Problem 2.6**: The answers should be σ_x = 130.16 MPa , σ_y= - 69.84 MPa

p. 171: **Problem 2.10**. Figure 2-36, D = 100 mm is missing.
p. 173: **Problem 2.15**: use Goodman equation.

p. 174: **Problem 2.18**: use Goodman equation. Answer: Fs = 1.75

Bending moment varies from 1,000,00 to 5,000,000 Nmm, not 1,040,000.

p. 175: **Problem 2.23**. Figure 2-49, the fillet radius r=0.5 and the width of the smaller side = 2 inch.
p. 177: **Problem 2.27.** *Ans.* (a) F_s=3.46; (b) F_s=3.81
p. 177: **Problem 2.30.** *Ans.* (a) F_s=4.02
p.192: Example 3.3, By Eqs.(2) and (5)

T= τJ/r = ……., the letter l is an error.

p. 207: Example 3-8, d_{shaft}=3-7/16 , not 3-7/6.

p. 209: Figure 3-11, the left side figure is Sled-Runner Keyway, the right figure is Profile Keyway.

p. 210: Figure 3-12, Remove "Sled-Runner Keyway", "Profile Keyway" from the figure.

p. 212: First calculation: τ = 7,000/1,841, not 1, 841. The comma is wrong.

p. 221: Example 3-15, In Eq. (16) calculation under f = …. = 11.8 0 cycles/sec Under the radical the "W 80x0.7922$^{2"}$, W should be deleted

p.222: Example 3.16: Find the deflection for point A. This is a miss spelling for deflection.

p. 227: EQ. (21) The numerator should be a letter τ, not t.

p. 229: Module 3-2 , last line: it should be b/c and not d/c
p.. 233 : third line below EQ. (28), is designated by a not α.

p. 234: Example 3-19 Solution (a):

By EQ. (29): we have θ_1 = ……. =0.000729 Rad./length =0.0418 ^{o}/length.

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in the above equation the value of 6.565^2 should be 6.562^2.

p. 259: **Problem 3.84**: In Figure 3-83 the distance of the 300 lb load is 20 inches not 25 inches. The force for the bearing R_3 should be R_3 not R_2.